

# Problem 1

PS 10

Mery Gandyk

The cylinder is infinite, thus we can assume due to symmetry in  $\theta$ , and translational symmetry in  $z$  that velocity field is:



$$\vec{u} = u_r(r) \hat{e}_r$$

a) div in cylindrical coordinates:

$$d(\vec{A} \lrcorner \Omega) = (\text{div } \vec{A}) \Omega$$

$$\begin{aligned} \Omega &= \sqrt{\det g} \, dr \wedge d\theta \wedge dz = \\ &= r \, dr \wedge d\theta \wedge dz \end{aligned}$$

So:

$$d(\vec{A} \lrcorner \Omega) = \left[ \partial_r (r A_r) + r \partial_\theta (A_\theta) + r \partial_z (A_z) \right] \Omega$$

$$\text{So } \text{div } \vec{A} = \frac{1}{r} \partial_r (r A_r) + \partial_\theta (A_\theta) + \partial_z (A_z)$$

but if we remember that  $\vec{A} = A_r^r \hat{e}_r + A_\theta^\theta \hat{e}_\theta + A_z^z \hat{e}_z = A_r^r \hat{e}_r + A_\theta^r \hat{e}_r + A_\theta^\theta \hat{e}_\theta + A_z^r \hat{e}_r + A_z^\theta \hat{e}_\theta + A_z^z \hat{e}_z$

We have

$$\text{div } \vec{A} = \frac{1}{r} \partial_r (r A_r^r) + \frac{1}{r} \partial_\theta (r A_\theta^r) + \partial_z A_z^z$$

So for our field  $\vec{u} = u_r(r) \hat{e}_r$  we have:

$$\vec{\nabla} \cdot \vec{u} = \frac{1}{r} \partial_r (r u_r)$$

b) To find deformation field we need to solve Navier-Cauchy:

$$\vec{f} + \mu \vec{\nabla}^2 \vec{u} + (\mu + \lambda) \vec{\nabla} (\vec{\nabla} \cdot \vec{u}) = 0$$

but for our field  $\vec{\nabla} \times \vec{u} = 0 \Rightarrow \vec{\nabla}^2 \vec{u} = \vec{\nabla} (\vec{\nabla} \cdot \vec{u})$

and we have body force due to rotation  $\vec{f} = \rho \omega^2 r \hat{e}_r$   
centrifugal force!

$$S_z \quad (2\mu + \lambda) \nabla (\nabla \cdot \bar{u}) = \rho \omega^2 r \hat{e}_r$$

$$\text{Thus} \quad (2\mu + \lambda) \nabla \left[ \frac{1}{r} S_r (r u_r) \right] = \rho \omega^2 r$$

$$S_r (r u_r) = -\frac{\rho \omega^2}{2\mu + \lambda} \frac{r^3}{2} + \overset{\text{const}}{2Ar}$$

$$u_r(r) = -\frac{\rho \omega^2}{2\mu + \lambda} \frac{r^3}{8} + Ar + \frac{B}{r}$$

but for a cylinder we also have boundary conditions:  
we considered rotating frame of reference with the cylinder,  
so:  $u_r(0) = 0$  + finite in the middle  $\Rightarrow B = 0$

And <sup>with</sup> outer edge of cylinder should be stress free:

$$\hat{T} = 2\mu \hat{E} + \lambda \text{Tr} \hat{E} \mathbb{1}, \text{ where } \hat{E} = (\nabla \bar{u})^s = \begin{bmatrix} S_r u_r \\ \frac{u_r}{r} \end{bmatrix}^s \quad \leftarrow \text{we can drop } z \text{ dependence}$$

$$\hat{E} = \begin{bmatrix} -3\beta r^2 + A \\ -\beta r^2 + A \end{bmatrix}$$

Similarly check:

$$\text{div } \bar{u} = -4\beta r^2 + 2A$$

$$\text{Tr } \hat{E} = -4\beta r^2 + 2A \quad \checkmark$$

$$S_z \quad \hat{T} = \text{diag} \left( -\beta r^2 (\delta_\mu + 4\lambda) + A(2\mu + 2\lambda), -\beta r^2 (2\mu + 4\lambda) + A(2\mu + 2\lambda) \right)$$

$$\text{and} \quad T_{rr}(r=R) = 0 \Rightarrow -\beta R^2 (\delta_\mu + 4\lambda) + A(2\mu + 2\lambda) = 0$$

$$\Rightarrow A = +\beta R^2 \frac{3\mu + 2\lambda}{\mu + \lambda} = +\beta R^2 \frac{3 + \frac{4\nu}{1-2\nu}}{1 + \frac{2\nu}{1-2\nu}} = +\beta R^2 (3 - 2\nu)$$

# Problem 1 cd

Jerry Gondszyk

So we finally get deformation field:

$$u_r = -\beta r^3 + Ar = -\beta r^3 + \beta R^2 r (3-2\nu) =$$

$$= \beta r R^2 \left( 3-2\nu - \frac{r^2}{R^2} \right) = \frac{\beta \omega^2 r R^3}{8(2\mu+\lambda)} \left( 3-2\nu - \frac{r^2}{R^2} \right)$$

$\text{sgn } u_r = \text{sgn} \left( 3-2\nu - \frac{r^2}{R^2} \right) = 1 \leftarrow$  the cylinder expands everywhere

(c) Now let's decompose the ~~stress~~ <sup>strain</sup> tensor:

~~$$T_{rr} = \beta r^2 (6\mu+\lambda) + A(2\mu+2\lambda) = \dots$$~~

$$E_{rr} = \nu_{,r} u_r = \beta R^2 \left( 3-2\nu - \frac{3r^2}{R^2} \right)$$

$$E_{\theta\theta} = \frac{u_r}{r} = \beta R^2 \left( 3-2\nu - \frac{r^2}{R^2} \right) > 0 \text{ everywhere.}$$

So strain in  $\theta$  corresponds to expansion everywhere on the other hand

$$E_{rr} = 0 \Leftrightarrow r_0 = R \sqrt{1 - \frac{2}{3}\nu} \leftarrow \text{exists for } \nu > 0 \text{ what about } \nu < 0?$$

Thus strain in  $\hat{e}_r$  corresponds to extension close to the center and to contraction further away.

d) If we consider breakage we observe the

$$(E_{rr})_{\max} = (E_{\theta\theta})_{\max} = \beta R^2 (3-2\nu) \leftarrow \text{at the center } r=0 \text{ we have biggest strains}$$

And in small neighbourhood  $E_{\theta\theta} > E_{rr}$  So breakage will take place in the middle along plane  $(r, z)$  for some  $\theta$ :



b\*) Distribution of stress:

$$T_{rr} = -2\beta r^2(3\mu + 2\lambda) + 2\beta R^2(3-2\nu)(\mu + \lambda) =$$
$$= 2\beta(3\mu + 2\lambda)(R^2 - r^2)$$

$$T_{\theta\theta} = -2\beta r^2(\mu + 2\lambda) + 2\beta R^2(3\mu + 2\lambda) =$$
$$= 2\beta[\mu(3R^2 - r^2) + 2\lambda(R^2 - r^2)]$$

Remarks •  $\nabla \times \bar{u} = \nabla \times (u_r \hat{e}_r) = \underbrace{(\nabla u_r)}_{f \hat{e}_r} \times \hat{e}_r + u_r \underbrace{\nabla \times \hat{e}_r}_0 = 0$

• For breakage it will certainly start at  $r=0$  but the difference between  $E_{rr}$  and  $E_{\theta\theta}$  is  $\sim \left(\frac{r}{R}\right)^2$  so it is not necessary for it to go in ~~the~~  $(r, z)$  plane  
- I am not sure

Zad 2 Szymon Kłopotnicki 440454

$$\begin{pmatrix} u_x \\ u_y \end{pmatrix} = \begin{pmatrix} \varepsilon & -\gamma \\ \gamma & -\varepsilon \end{pmatrix} \begin{pmatrix} x \\ y \end{pmatrix} = \begin{pmatrix} \varepsilon x - \gamma y \\ \gamma x - \varepsilon y \end{pmatrix}$$

$$\vec{\nabla} \cdot \vec{u} = \frac{\partial u_x}{\partial x} + \frac{\partial u_y}{\partial y} = \frac{\partial}{\partial x}(\varepsilon x - \gamma y) + \frac{\partial}{\partial y}(\gamma x - \varepsilon y) = \varepsilon - \varepsilon = 0 \quad \checkmark$$

$$\vec{u} = \vec{\nabla} \psi = \begin{pmatrix} \frac{\partial \psi}{\partial x} \\ \frac{\partial \psi}{\partial y} \\ 0 \end{pmatrix} = \begin{pmatrix} u_x \\ u_y \\ 0 \end{pmatrix}$$

$$1) \frac{\partial \psi}{\partial y} = \varepsilon x - \gamma y \quad 2) \frac{\partial \psi}{\partial x} = \gamma x - \varepsilon y$$

$$1) \psi = \varepsilon x y - \frac{1}{2} \gamma y^2 + f(x)$$

$$2) \psi = \varepsilon x y - \frac{1}{2} \gamma x^2 + f(y)$$

$$\Rightarrow \psi = -\frac{1}{2} \gamma (x^2 + y^2) + \varepsilon x y \quad (+ \text{const}) \quad (*)$$

Checking the formula:

$$(\varepsilon - \gamma)(x^2 + 2xy + y^2) - (\varepsilon + \gamma)(x^2 - 2xy + y^2) = C$$

$$\varepsilon(2xy + 2xy) - \gamma(2x^2 + 2y^2) = C$$

$$4\varepsilon xy - 2\gamma(x^2 + y^2) = C$$

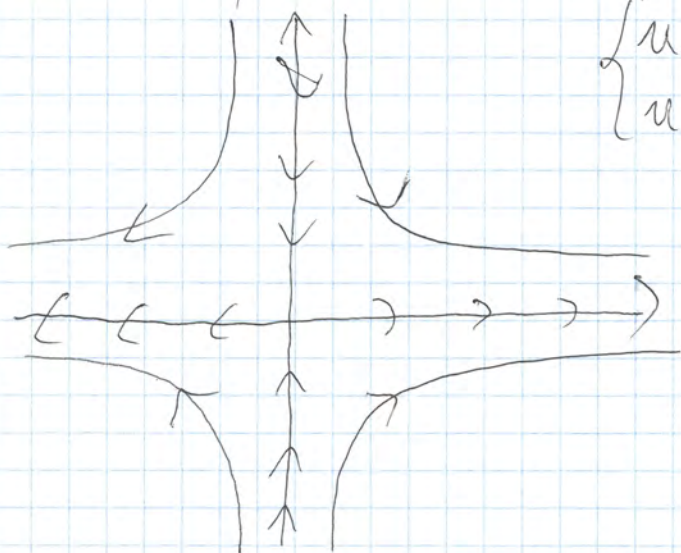
$$\varepsilon xy - \frac{1}{2} \gamma (x^2 + y^2) = \frac{C}{4} = \text{const} \quad \checkmark$$

Is the same as (\*), so it's OK

Streamlines  $\Rightarrow \psi = \text{const} = C$

$$1) \varepsilon = 1, \gamma = 0 \Rightarrow (x+y)^2 - (x-y)^2 = \text{const}$$

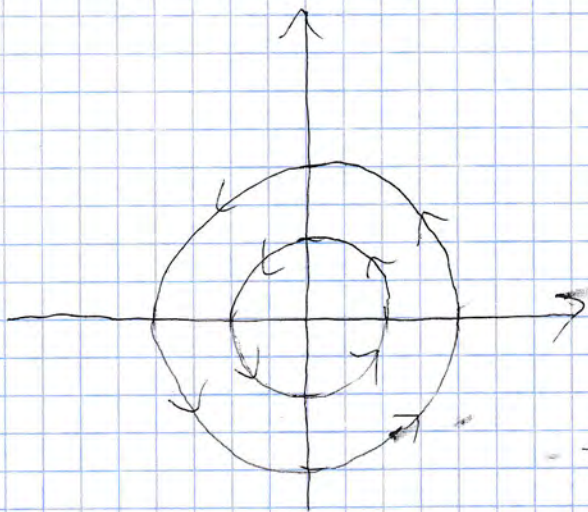
$$\Rightarrow xy = \text{const}$$



$$\begin{cases} u_x = x \\ u_y = -y \end{cases}$$

further away there  
are higher speeds,  
 $\vec{u}(0,0) = \vec{0}$

$$2) \quad \varepsilon = 0, \quad \gamma = 1 \Rightarrow x^2 + y^2 = \text{const}$$



$$\begin{cases} u_x = -y \\ u_y = x \end{cases}$$

Vortex!

(I think this is "bathtub vortex")

• further away - higher speeds  $\sim r$   
 where  $r = \text{radius}$

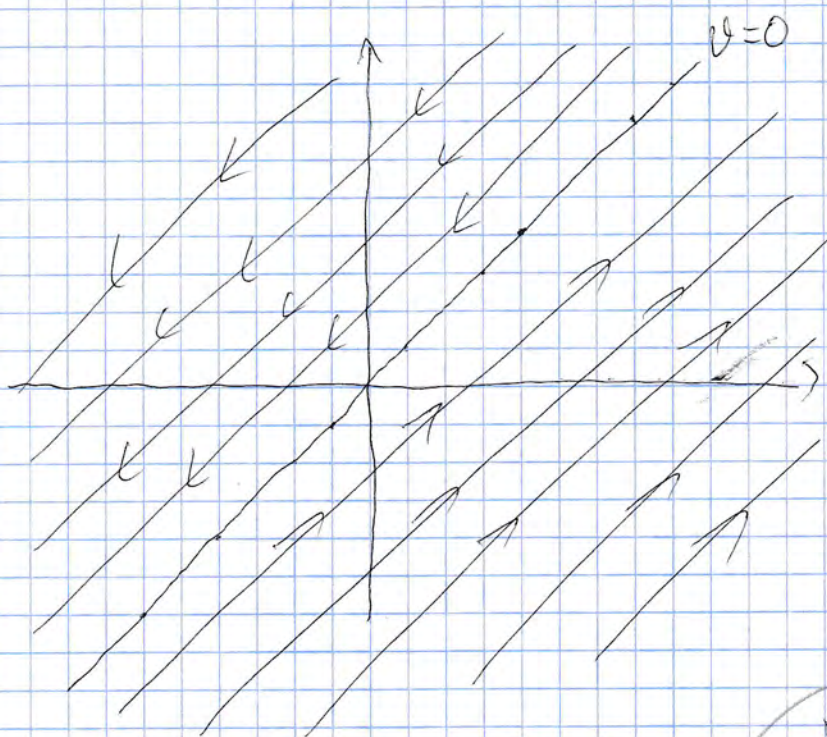
$$3) \quad \varepsilon = 1, \quad \gamma = 1 \Rightarrow$$

$$-\frac{1}{2}(x^2 + y^2) + xy = \text{const} \quad | \cdot (-2)$$

$$x^2 + y^2 - 2xy = \text{const}$$

$$(x - y)^2 = \text{const}$$

$$y = x \pm \text{const}$$



$$\begin{cases} u_x = x - y \\ u_y = x - y \end{cases}$$

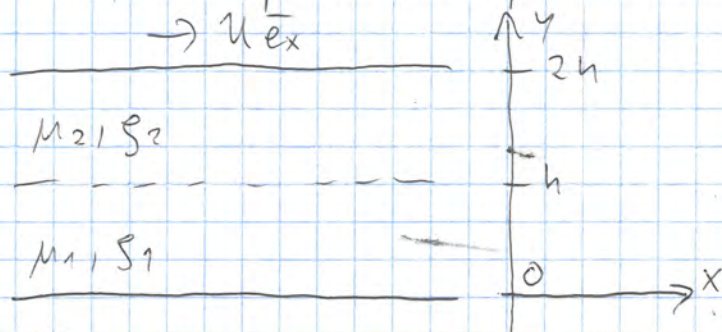
$\psi = 0$  on line  $x = y$

for from line  $x = y$   
 we get higher speeds



Good job

Zad 4 Symon Klopinski 440454



$$\vec{v}(x, y) = v_x(y) \vec{e}_x$$

$$\frac{\partial \vec{v}}{\partial t} + (\vec{v} \cdot \nabla) \vec{v} = -\frac{1}{\rho} \nabla \bar{p} + \frac{\mu}{\rho} \nabla^2 \vec{v} + \vec{f}$$

no time dependence, unidirectional flow ( $\vec{v} = v_x(y) \vec{e}_x$ )

no body forces, no pressure gradient mentioned

So, we are left with  $0 = \frac{\mu}{\rho} \nabla^2 \vec{v}$  ✓

We can multiply by  $\frac{\rho}{\mu}$ , and we get:

$$\nabla^2 \vec{v} = 0 \Rightarrow \frac{\partial^2 v_x}{\partial y^2} = 0 \quad \checkmark$$

$v_x(y) = Ay + B$  We have 2 fluids:

$$v_x^1(y) = Ay + B \quad v_x^2(y) = Cy + D$$

Boundary conditions:

1)  $v_x^1(0) = 0$

3)  $v_x^1(h) = v_x^2(h)$  ✓

2)  $\bar{v}_x^2(2h) = u \vec{e}_x$

4)  $\mu_1 \frac{\partial v_x^1}{\partial y} \Big|_{y=h} = \mu_2 \frac{\partial v_x^2}{\partial y} \Big|_{y=h}$

1)  $\Rightarrow v_x^1 = Ay$  ;  $B = 0$  ✓

2)  $\Rightarrow 2hC + D = u \Rightarrow D = u - 2hC$  ✓

3)  $\Rightarrow Ah = Ch + u - 2Ch$

4)  $\Rightarrow \mu_1 A = \mu_2 C \Rightarrow A = \frac{\mu_2}{\mu_1} C$

3)  $\Rightarrow \frac{\mu_2}{\mu_1} Ch = Ch - 2Ch + u \Rightarrow C = \frac{u}{h(\frac{\mu_2}{\mu_1} + 1)}$  ✓

$\Rightarrow A = \frac{u}{h(\frac{\mu_2}{\mu_1} + 1)}$  ✓

So we get:

On the next page

$$v_x^1(y) = \frac{u}{h \left( \frac{\mu_1}{\mu_2} + 1 \right)} y$$

$$v_x^2(y) = \frac{u}{h \left( \frac{\mu_2}{\mu_1} + 1 \right)} (y - 2h) + u$$

All the boundary conditions are satisfied (✓ checked)

no dependence on  $z$  in fact.

Stress tensor:

$$\hat{T} = 2\mu \hat{D} \quad \hat{D} = \frac{1}{2} (\nabla \vec{v})^S$$

$$\hat{D} = \begin{pmatrix} 0 & \frac{1}{2} \frac{\partial v_x}{\partial y} \\ \frac{1}{2} \frac{\partial v_x}{\partial y} & 0 \end{pmatrix}$$

$$\frac{\partial v_x^1}{\partial y} = \frac{u}{h \left( \frac{\mu_1}{\mu_2} + 1 \right)} \quad \frac{\partial v_x^2}{\partial y} = \frac{u}{h \left( \frac{\mu_2}{\mu_1} + 1 \right)}$$

$$t_{xy}^1 = 2\mu_1 \cdot \frac{1}{2} \frac{u}{h \left( \frac{\mu_1}{\mu_2} + 1 \right)} = \frac{u}{h \left( \frac{1}{\mu_1} + \frac{1}{\mu_2} \right)} \quad \checkmark$$

$$t_{xy}^2 = 2\mu_2 \cdot \frac{1}{2} \frac{u}{h \left( \frac{\mu_2}{\mu_1} + 1 \right)} = \frac{u}{h \left( \frac{1}{\mu_1} + \frac{1}{\mu_2} \right)} \quad \checkmark$$

In fact,  $t_{xy}^1 = t_{xy}^2 = \text{const}$  in whole fluid.

10/10 bravo



Szymon Michniak 421432 Problem 3 (10) PS

The bubble experiences a force due to displaced amount of fluid (if  $\vec{a}$  would be gravity, this would be buoyancy force).

$$F_1 = V_r \rho_f \vec{a}$$

Another force comes from the drag the bubble experiences in the ~~unsteady flow~~ unsteady flow. The effective mass for a ball is given by  $M_{\text{eff}} = M + \frac{M'}{2}$  where  $M'$  is the mass of displaced fluid. This ~~is~~ drag comes from the difference in accelerations and is connected with the addition to effective mass.

$$F_2 = -\frac{M'}{2} (a_r - a) = -\frac{V_r \rho_f}{2} (a_r - a)$$

Both forces contribute to the total force acting on the bubble

$$m_r a_r = F_1 + F_2 = V_r \rho_f a - \frac{V_r \rho_f}{2} (a_r - a) = \frac{3}{2} V_r \rho_f a - \frac{V_r \rho_f}{2} a_r$$

$\leftarrow V_r \rho_f a_r$

$$a_r \left( \rho_r + \frac{\rho_f}{2} \right) = \frac{3}{2} \rho_f a$$

$\swarrow \rho_r \ll \rho_f$

$$a_r = \frac{3 \rho_f}{\rho_f + 2 \rho_r} a = \frac{3}{1 + 2 \frac{\rho_r}{\rho_f}} a \approx 3a$$